# 6.003: Signals and Systems

**Frequency Response** 

October 6, 2011

#### Review

Last time, we saw how a linear, time-invariant (LTI) system can be characterized by its unit-sample/impulse response.

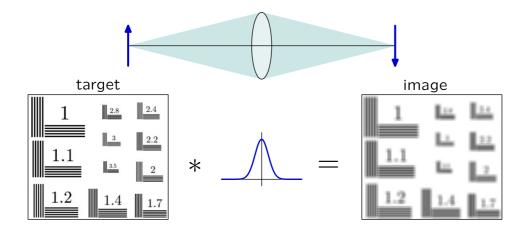
DT: 
$$y[n] = (x * h)[n] = \sum_{k=-\infty}^{\infty} x[k]h[n-k]$$

CT: 
$$y(t) = (x * h)(t) = \int_{-\infty}^{\infty} x(\tau)h(t - \tau)d\tau$$

Characterizing a system by its unit-sample/impulse response is especially insightful for some systems.

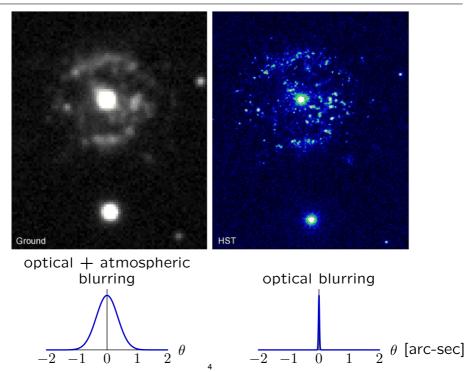
## Microscope

Blurring can be represented by convolving the image with the optical "point-spread-function" (3D impulse response).



Blurring is inversely related to the diameter of the lens.

#### Hubble Space Telescope



### **Frequency Response**

Today we will investigate a different way to characterize a system: the **frequency response**.

Many systems are naturally described by their responses to sinusoids.

Example: audio systems

# **Check Yourself**

How were frequencies modified in following music clips?				
HF: high frequencies		↑: increased		
LF: low frequencies		↓: decreased		
	-11 1			
	clip 1	clip 2		
1.	HF↑	HF↓		
2.	LF↑	LF↓		
3.	HF↑	LF↓		
4.	LF↑	HF↓		
5.	none of	the above		

# **Check Yourself**

original clip 1:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 1:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 2:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 2:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 2:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none

clip	1	clip	2

- 1.  $\mathsf{HF}\uparrow$   $\mathsf{HF}\downarrow$
- 2.  $LF\uparrow$   $LF\downarrow$
- 3. HF↑ LF↓
- 4. LF↑ HF↓
- 5. none of the above

# **Check Yourself**

original clip 1:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 1:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 2:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 2:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none original clip 2:  $HF\uparrow HF\downarrow LF\uparrow LF\downarrow$  none

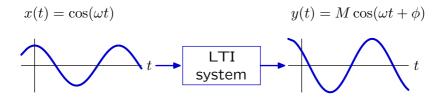
	clip 1	clip 2
1.	HF↑	HF↓
2.	LF↑	LF↓

- 3. HF↑ LF↓
- 4.  $LF\uparrow$   $HF\downarrow$
- 5. none of the above

#### **Frequency Response Preview**

If the input to a linear, time-invariant system is an eternal sinusoid, then the output is also an eternal sinusoid:

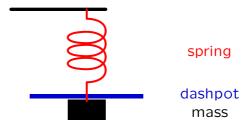
- same frequency
- possibly different amplitude, and
- possibly different phase angle.



The **frequency response** is a plot of the magnitude M and angle  $\phi$  as a function of frequency  $\omega$ .

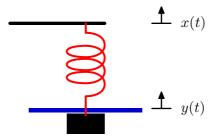
# Example

Mass, spring, and dashpot system.



# Demonstration

Measure the frequency response of a mass, spring, dashpot system.



## **Frequency Response**

Calculate the frequency response.

Methods

• solve differential equation

 $\rightarrow$  find particular solution for  $x(t)=\cos\omega_0 t$ 

• find impulse response of system

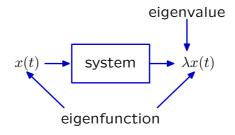
 $\rightarrow$  convolve with  $x(t) = \cos \omega_0 t$ 

New method

• use eigenfunctions and eigenvalues

# Eigenfunctions

If the output signal is a scalar multiple of the input signal, we refer to the signal as an eigenfunction and the multiplier as the eigenvalue.



Consider the system described by  $\dot{y}(t) + 2y(t) = x(t)$ . Determine if each of the following functions is an eigenfunction of this system. If it is, find its eigenvalue.

- 1.  $e^{-t}$  for all time
- 2.  $e^t$  for all time
- 3.  $e^{jt}$  for all time
- 4.  $\cos(t)$  for all time
- 5. u(t) for all time

#### **Check Yourself: Eigenfunctions**

$$\dot{y}(t) + 2y(t) = x(t)$$

1. 
$$e^{-t}$$
:  $-\lambda e^{-t} + 2\lambda e^{-t} = e^{-t} \rightarrow \lambda = 1$ 

2. 
$$e^t$$
:  $\lambda e^t + 2\lambda e^t = e^t \to \lambda = \frac{1}{3}$ 

3. 
$$e^{jt}$$
:  $j\lambda e^{jt} + 2\lambda e^{jt} = e^{jt} \rightarrow \lambda = \frac{1}{j+2}$ 

4.  $\cos t$ :  $-\lambda \sin t + 2\lambda \cos t = \cos t \rightarrow$  not possible!

5.  $u(t): \lambda \delta(t) + 2\lambda u(t) = u(t) \rightarrow \text{ not possible!}$ 

Consider the system described by  $\dot{y}(t) + 2y(t) = x(t)$ . Determine if each of the following functions is an eigenfunction of this system. If it is, find its eigenvalue.

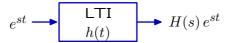
1.  $e^{-t}$  for all time  $\sqrt{\lambda} = 1$ 2.  $e^{t}$  for all time  $\sqrt{\lambda} = \frac{1}{3}$ 3.  $e^{jt}$  for all time  $\sqrt{\lambda} = \frac{1}{j+2}$ 4.  $\cos(t)$  for all time X 5. u(t) for all time X

#### **Complex Exponentials**

Complex exponentials are eigenfunctions of LTI systems.

If  $x(t) = e^{st}$  and h(t) is the impulse response then

$$y(t) = (h * x)(t) = \int_{-\infty}^{\infty} h(\tau) e^{s(t-\tau)} d\tau = e^{st} \int_{-\infty}^{\infty} h(\tau) e^{-s\tau} d\tau = H(s) e^{st}$$



Eternal sinusoids are sums of complex exponentials.

$$\cos\omega_0 t = \frac{1}{2} \left( e^{j\omega_0 t} + e^{-j\omega_0 t} \right)$$

Furthermore, the eigenvalue associated with  $e^{st}$  is H(s)!

## **Rational System Functions**

Eigenvalues are particularly easy to evaluate for systems represented by linear differential equations with constant coefficients.

Then the system function is a ratio of polynomials in s.

Example:

$$\ddot{y}(t) + 3\dot{y}(t) + 4y(t) = 2\ddot{x}(t) + 7\dot{x}(t) + 8x(t)$$

Then

$$H(s) = \frac{2s^2 + 7s + 8}{s^2 + 3s + 4} \equiv \frac{N(s)}{D(s)}$$

The value of H(s) at a point  $s = s_0$  can be determined graphically using vectorial analysis.

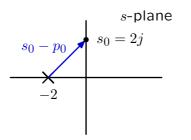
Factor the numerator and denominator of the system function to make poles and zeros explicit.

$$H(s_0) = K \frac{(s_0 - z_0)(s_0 - z_1)(s_0 - z_2)\cdots}{(s_0 - p_0)(s_0 - p_1)(s_0 - p_2)\cdots}$$
so plane
so so so plane
so so so plane

Each factor in the numerator/denominator corresponds to a vector from a zero/pole (here  $z_0$ ) to  $s_0$ , the point of interest in the *s*-plane.

Example: Find the response of the system described by  $H(s) = \frac{1}{s+2} \label{eq:H}$ 

to the input  $x(t) = e^{2jt}$  (for all time).



The denominator of  $H(s)|_{s=2j}$  is 2j+2, a vector with length  $2\sqrt{2}$  and angle  $\pi/4$ . Therefore, the response of the system is

$$y(t) = H(2j)e^{2jt} = \frac{1}{2\sqrt{2}}e^{-\frac{j\pi}{4}}e^{2jt}$$

The value of H(s) at a point  $s = s_0$  can be determined by combining the contributions of the vectors associated with each of the poles and zeros.

$$H(s_0) = K \frac{(s_0 - z_0)(s_0 - z_1)(s_0 - z_2)\cdots}{(s_0 - p_0)(s_0 - p_1)(s_0 - p_2)\cdots}$$

The magnitude is determined by the product of the magnitudes.  $|H(s_0)| = |K| \frac{|(s_0 - z_0)||(s_0 - z_1)||(s_0 - z_2)|\cdots}{|(s_0 - p_0)||(s_0 - p_1)||(s_0 - p_2)|\cdots}$ 

The angle is determined by the sum of the angles.

$$\angle H(s_0) = \angle K + \angle (s_0 - z_0) + \angle (s_0 - z_1) + \dots - \angle (s_0 - p_0) - \angle (s_0 - p_1) - \dots$$

#### **Frequency Response**

Response to eternal sinusoids.

Let 
$$x(t) = \cos \omega_0 t$$
 (for all time). Then  $x(t) = \frac{1}{2} \left( e^{j\omega_0 t} + e^{-j\omega_0 t} \right)$ 

and the response to a sum is the sum of the responses.

$$y(t) = \frac{1}{2} \left( H(j\omega_0) e^{j\omega_0 t} + H(-j\omega_0) e^{-j\omega_0 t} \right)$$

#### **Conjugate Symmetry**

The complex conjugate of  $H(j\omega)$  is  $H(-j\omega)$ .

The system function is the Laplace transform of the impulse response:

$$H(s) = \int_{-\infty}^{\infty} h(t)e^{-st}dt$$

where h(t) is a real-valued function of t for physical systems.

$$H(j\omega) = \int_{-\infty}^{\infty} h(t)e^{-j\omega t}dt$$
$$H(-j\omega) = \int_{-\infty}^{\infty} h(t)e^{j\omega t}dt \equiv \left(H(j\omega)\right)^{*}$$

#### **Frequency Response**

Response to eternal sinusoids.

Let 
$$x(t) = \cos \omega_0 t$$
 (for all time), which can be written as  $x(t) = \frac{1}{2} \left( e^{j\omega_0 t} + e^{-j\omega_0 t} \right)$ 

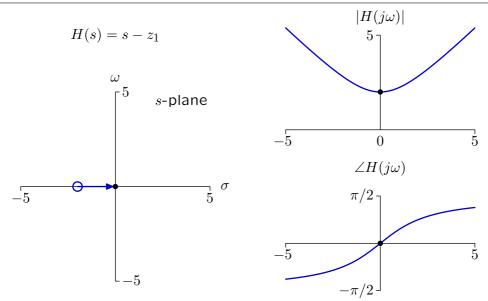
The response to a sum is the sum of the responses,

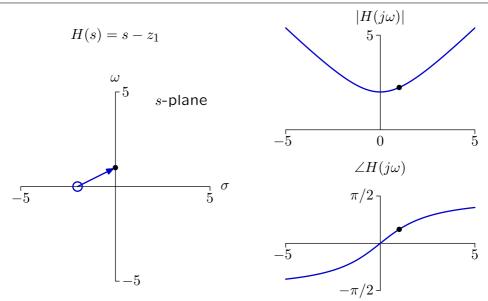
$$y(t) = \frac{1}{2} \left( H(j\omega_0)e^{j\omega_0 t} + H(-j\omega_0)e^{-j\omega_0 t} \right)$$
$$= \operatorname{Re} \left\{ H(j\omega_0)e^{j\omega_0 t} \right\}$$
$$= \operatorname{Re} \left\{ |H(j\omega_0)|e^{j\angle H(j\omega_0)}e^{j\omega_0 t} \right\}$$
$$= |H(j\omega_0)|\operatorname{Re} \left\{ e^{j\omega_0 t + j\angle H(j\omega_0)} \right\}$$
$$y(t) = |H(j\omega_0)| \cos \left( \omega_0 t + \angle H(j\omega_0) \right).$$

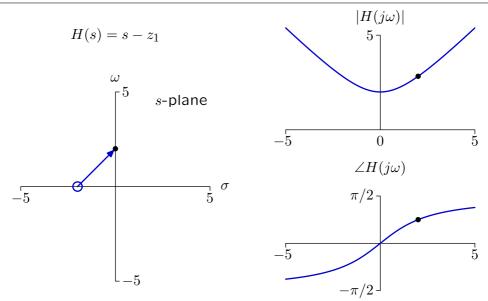
#### **Frequency Response**

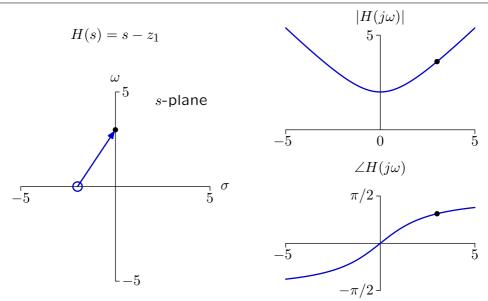
The magnitude and phase of the response of a system to an eternal cosine signal is the magnitude and phase of the system function evaluated at  $s = j\omega$ .

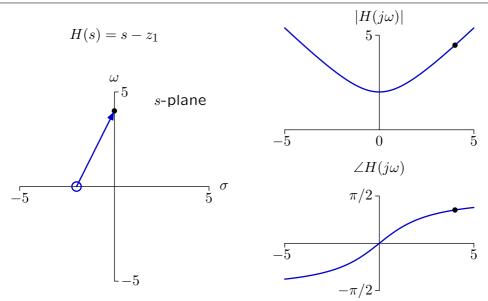
$$\cos(\omega t) \longrightarrow H(s) \longrightarrow |H(j\omega)| \cos(\omega t + \angle H(j\omega))$$

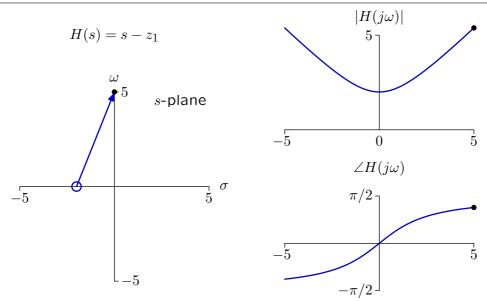


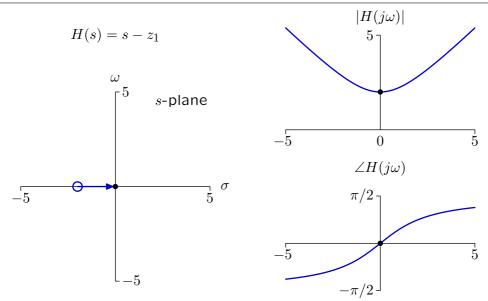


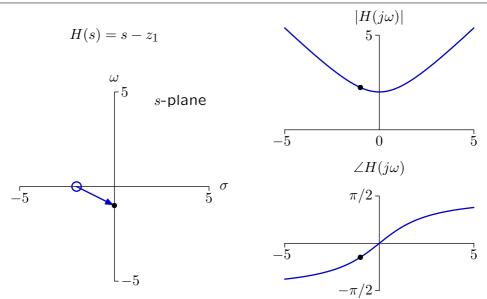


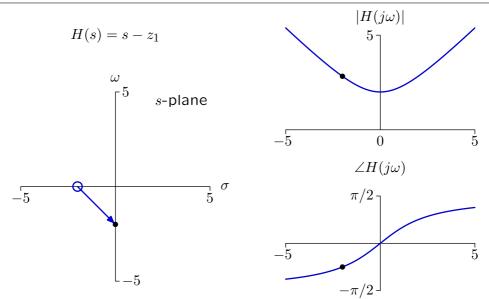


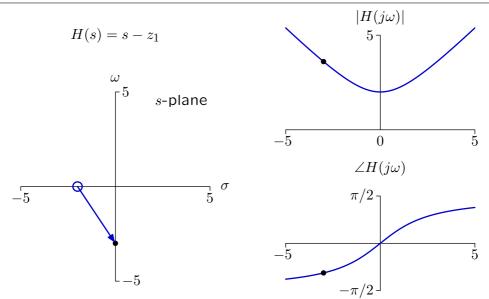


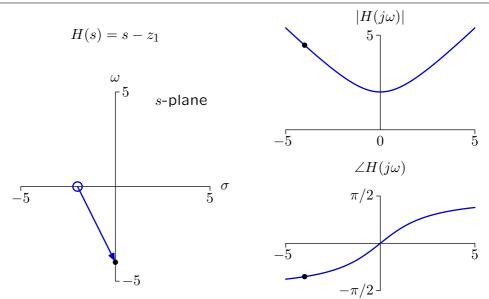


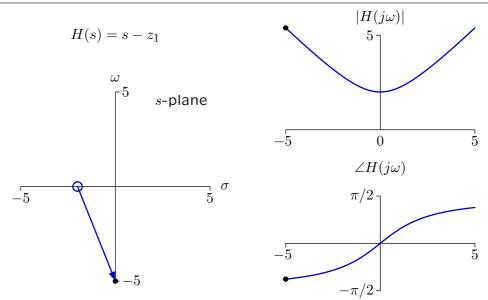


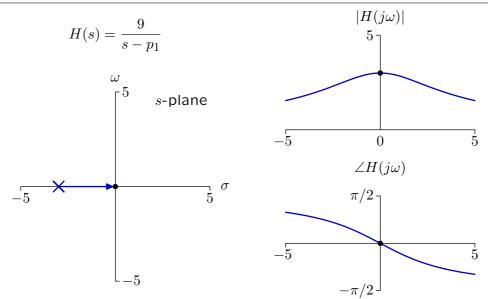


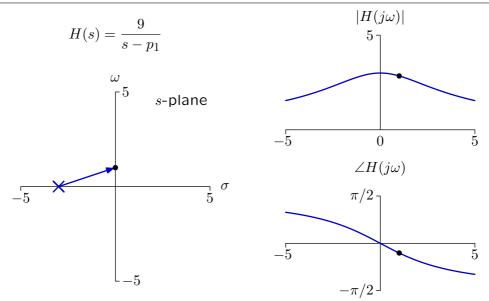


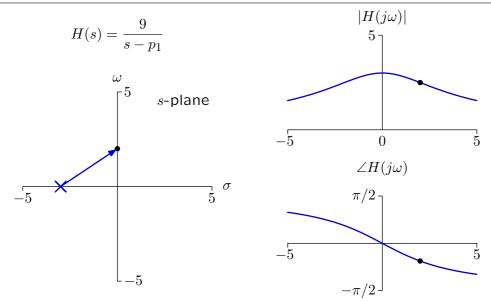


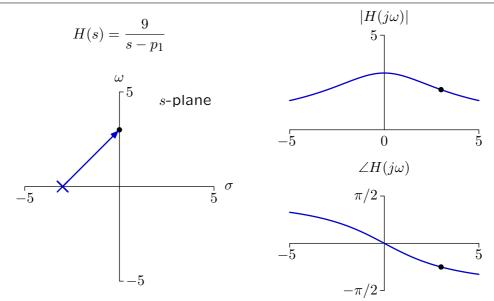


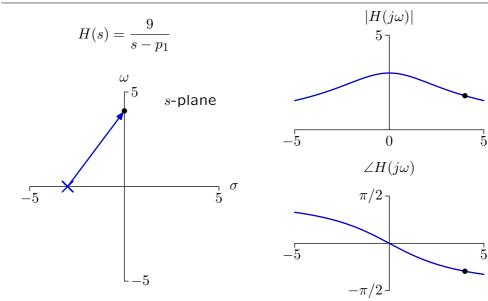




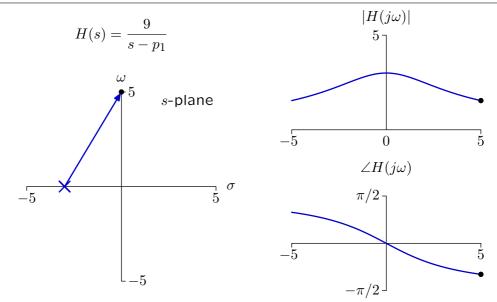


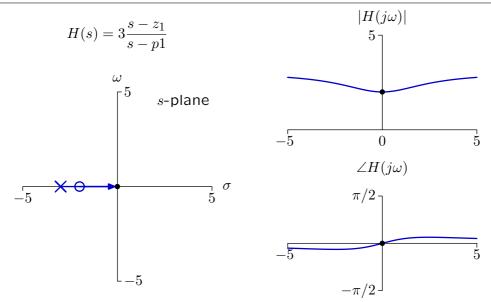


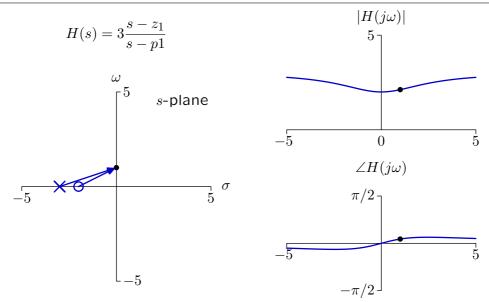


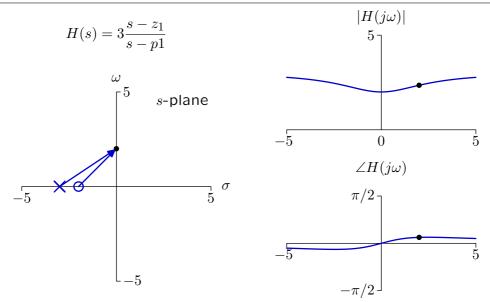


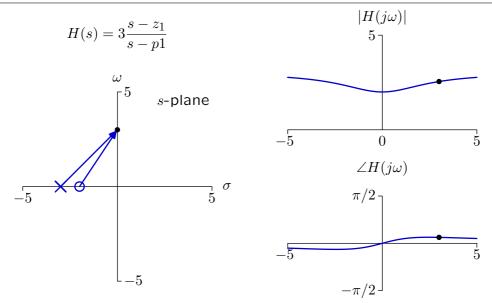
 $\dot{5}$ 

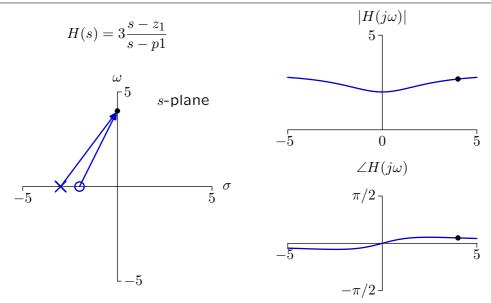


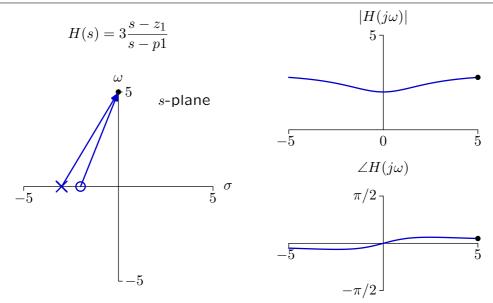




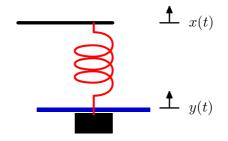




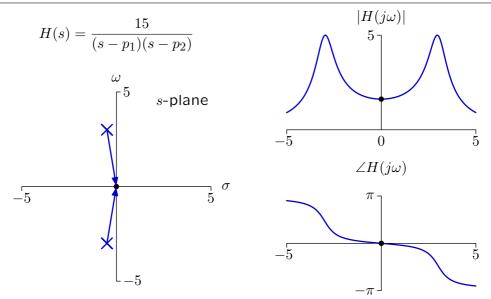


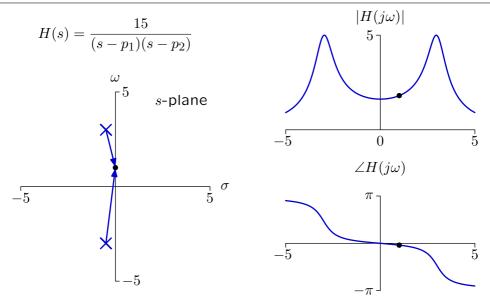


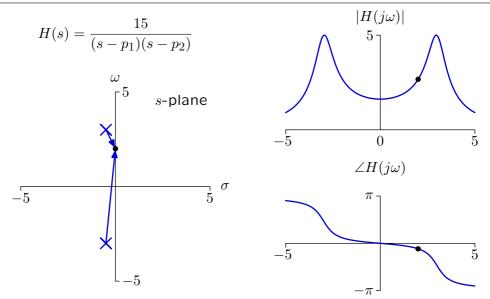
#### Example: Mass, Spring, and Dashpot

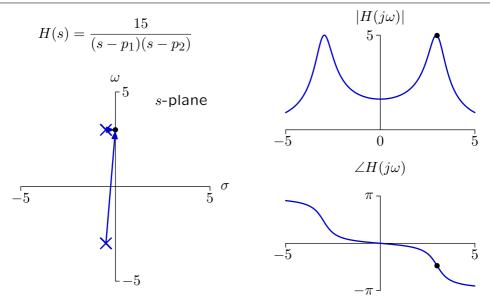


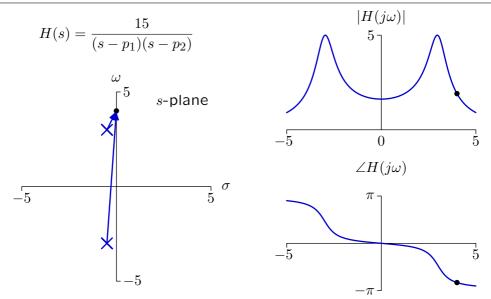
$$F = Ma = M\ddot{y}(t) = K(x(t) - y(t)) - B\dot{y}(t)$$
$$M\ddot{y}(t) + B\dot{y}(t) + Ky(t) = Kx(t)$$
$$(s^2M + sB + K) Y(s) = KX(s)$$
$$H(s) = \frac{K}{s^2M + sB + K}$$

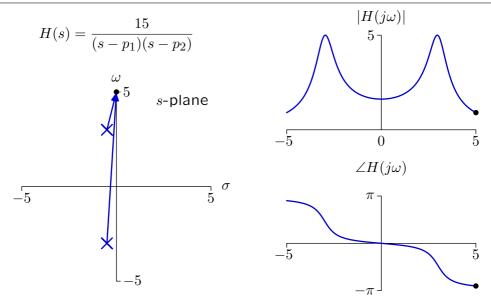










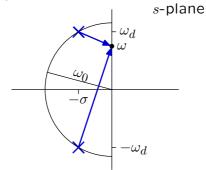


Consider the system represented by the following poles. s-plane  $\omega_d$  $\sigma$  $\omega_d$ Find the frequency  $\omega$  at which the magnitude of the response y(t) is greatest if  $x(t) = \cos \omega t$ . 1.  $\omega = \omega_d$ 2.  $\omega_d < \omega < \omega_0$ 

3.  $0 < \omega < \omega_d$  4. none of the above

# **Check Yourself: Frequency Response**

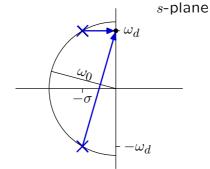
Analyze with vectors.



The product of the lengths is  $\left(\sqrt{(\omega+\omega_d)^2+\sigma^2}\right)\left(\sqrt{(\omega-\omega_d)^2+\sigma^2}\right)$ .

# **Check Yourself: Frequency Response**

Analyze with vectors.

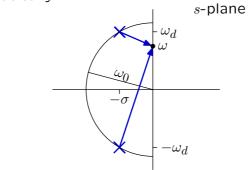


The product of the lengths is  $\left(\sqrt{(\omega + \omega_d)^2 + \sigma^2}\right) \left(\sqrt{(\omega - \omega_d)^2 + \sigma^2}\right)$ .

Decreasing  $\omega$  from  $\omega_d$  to  $\omega_d - \epsilon$  decreases the product since length of bottom vector decreases as  $\epsilon$  while length of top vector increases only as  $\epsilon^2$ .

# **Check Yourself: Frequency Response**

More mathematically ...



The product of the lengths is  $\left(\sqrt{(\omega + \omega_d)^2 + \sigma^2}\right) \left(\sqrt{(\omega - \omega_d)^2 + \sigma^2}\right)$ .

Maximum occurs where derivative of squared lengths is zero.

$$\frac{d}{d\omega}\left((\omega+\omega_d)^2+\sigma^2\right)\left((\omega-\omega_d)^2+\sigma^2\right)=0$$

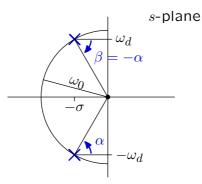
 $\rightarrow \quad \omega^2 = \omega_d^2 - \sigma^2 = \omega_0^2 - 2\sigma^2 \,.$ 

Consider the system represented by the following poles. s-plane  $\omega_d$  $-\omega_d$ Find the frequency  $\omega$  at which the magnitude of the response y(t) is greatest if  $x(t) = \cos \omega t$ . 3 1.  $\omega = \omega_d$ 2.  $\omega_d < \omega < \omega_0$ 3.  $0 < \omega < \omega_d$ 

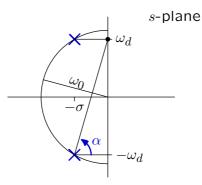
4. none of the above

Consider the system represented by the following poles. s-plane  $\omega_d$  $-\sigma$  $\omega_d$ Find the frequency  $\omega$  at which the phase of the response y(t) is  $-\pi/2$  if  $x(t) = \cos \omega t$ . 0.  $0 < \omega < \omega_d$ 1.  $\omega = \omega_d$ 2.  $\omega_d < \omega < \omega_0$ 4.  $\omega > \omega_0$ 3.  $\omega = \omega_0$ 5. none

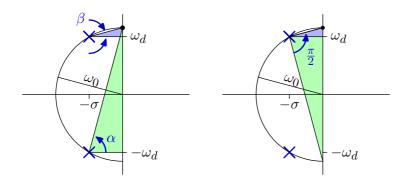
The phase is 0 when  $\omega = 0$ .



The phase is less than  $\pi/2$  when  $\omega = \omega_d$ .



The phase is  $-\pi/2$  at  $\omega = \omega_0$ .



Check result by evaluating the system function.

Substitute 
$$s = j\omega_0 = j\sqrt{\frac{K}{M}}$$
 into  

$$H(s) = \frac{K}{s^2M + sB + K} = \frac{K}{-\frac{K}{M}M + j\omega_0 B + K} = \frac{K}{j\omega_0 B}$$

The phase is  $-\frac{\pi}{2}$ .

Consider the system represented by the following poles. s-plane  $\omega_d$  $-\sigma$  $-\omega_d$ Find the frequency  $\omega$  at which the phase of the response y(t) is  $-\pi/2$  if  $x(t) = \cos \omega t$ . 3 0.  $0 < \omega < \omega_d$ 1.  $\omega = \omega_d$ 2.  $\omega_d < \omega < \omega_0$ 3.  $\omega = \omega_0$ 4.  $\omega > \omega_0$ 5. none

### Frequency Response: Summary

LTI systems can be characterized by responses to eternal sinusoids.

Many systems are naturally described by their frequency response.

- audio systems
- mass, spring, dashpot system

Frequency response is easy to calculate from the system function.

Frequency response lives on the  $j\omega$  axis of the Laplace transform.

MIT OpenCourseWare http://ocw.mit.edu

6.003 Signals and Systems Fall 2011

For information about citing these materials or our Terms of Use, visit: http://ocw.mit.edu/terms.